

Failure of brittle materials:

(A) characteristics of brittle material:

1. Do not have yield point (strength) ; σ - ϵ is smooth curve up to failure by fracture.
2. Comp. strength S_{uc} many times greater tensile S_{ut}
 $\Rightarrow S_{uc} > S_{ut}$
3. Ultimate torsional strength S_{us} (rupture modulus)
 $= S_{ut}$

(B) Failure theories (brittle materials):

1. Rankine theory
2. Coulomb-Mohr theory. (internal friction theory).
3. Modified Mohr theory.

1- Rankine @ previous ; only use S_{ut} & S_{uc} (Not yield)

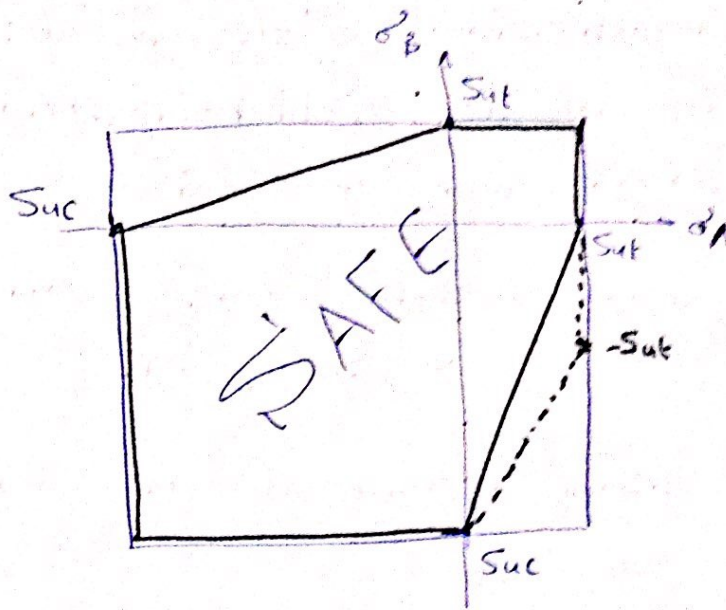
2- Coulomb-Mohr theory: " Failure (fracture) occurs when complex stress state produce a circle tangent to the envelope of ~~the~~ two test circles } tensile test for S_{ut} and compressive test for S_{uc} }

3- Modified Mohr theory: see - graphical sol.

Graphically

- see next page

- (1) Inside Zone \Rightarrow safe design ; outside \Rightarrow failure
- (2) @ 1st quadrant \Rightarrow Rankine coincides with Mohr-Coulomb theory
- (3) Exp. results coincides with Modified Mohr theory. especially @ 4th quadrant -



④ If $\sigma_1 > \sigma_2 > \sigma_3$
 and $\left\{ \begin{array}{l} \sigma_1 > 0 \\ \sigma_3 < 0 \end{array} \right\}$

Use theoretical eq:

$$\frac{\sigma_1}{S_{ut}} - \frac{\sigma_3}{S_{uc}} = 1$$

blue : Rankine theory
 black (solid) : Mohr-Coulomb theory
 black (dash) : Modified Mohr theory

Note: Use S_{uc} : + true value.

Example

Teach students how to use/get
 Yield & Ultimate stresses from
 Tables

Ex A small 6-mm diameter pin was designed of material near BS 300 Cast Iron ($S_{ut} = 293 \text{ MPa}$; $S_{uc} = 965 \text{ MPa}$)
 The pin take an axial compressive load of 3.5 kN combined with a torsional load of 9.8 N.m; Find safety factor for each of three failure theories.

Sol. Graphical sol.

step 1
 - axial comp. stress $\sigma_x = \frac{F}{A} = \frac{4F}{\pi d^2} = \frac{4 \times 3.5 \times 10^3}{\pi (6)^2} = -124 \text{ MPa}$
 - torsional shear stress $\tau_{xy} = \frac{16T}{\pi d^3} = \frac{16 \times 9.8 \times 10^3}{\pi (6)^3} = 231 \text{ MPa}$

step 2
 Find principal stresses \rightarrow equation
 \rightarrow Mohr Circle

$$\begin{cases} \sigma_1 = 177 \text{ MPa} \\ \sigma_2 = 0 \\ \sigma_3 = -301 \text{ MPa} \end{cases} \quad \text{For 2D } (\sigma_1 \text{ \& } \sigma_2) \text{ exist}$$

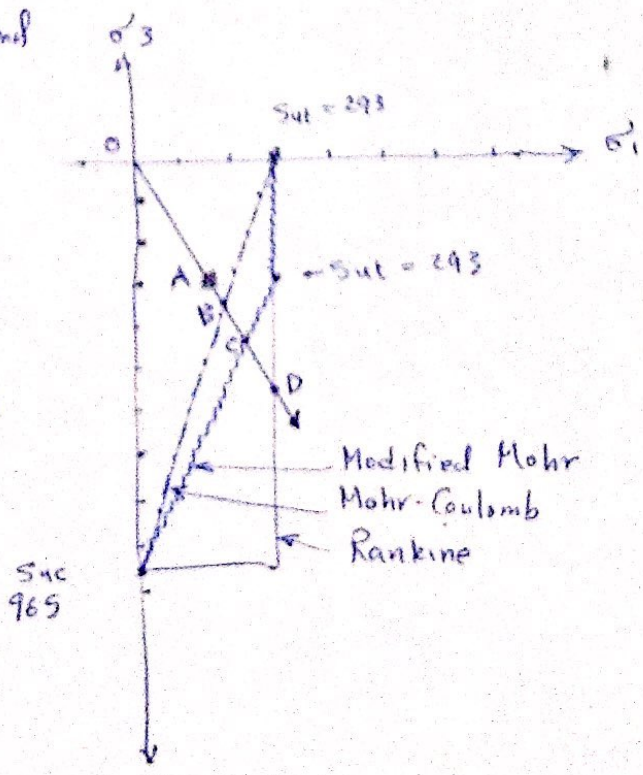
step 3 use graph paper (scale), plot 4th quadrant.

1. plot Rankine, Mohr-Coulomb and Modified Curves.

2. plot point A = (σ_1, σ_3)
 A : $(177, -301)$

3. Draw line OA through all failure curves OA, locate points B, C and D

4. $n = \frac{OB}{OA}$ (Rankine) = 1.6
 $n = \frac{OC}{OA}$ (Modified Mohr) = 1.35
 $n = \frac{OD}{OA}$ (Mohr-Coulomb) = 1.05



or use horizontal projections

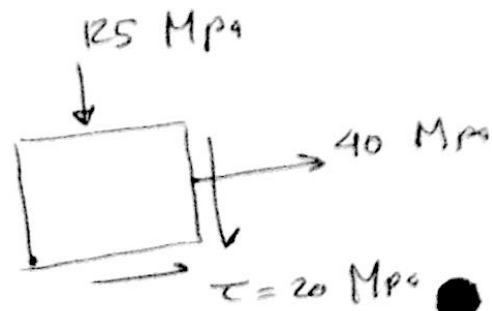
Ex #2

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CH 2

The stress state @ a point for it shown in Fig. for a material (same as previous example)

BS 300 Cast Iron ($S_{ut} = 293 \text{ MPa}$, $S_{uc} = 965 \text{ MPa}$)

find safety factor using (a) Rankine, (b) Coulomb-Mohr & (c) Modified Mohr theories.



Sol. Using Equations $\left\{ \begin{array}{l} \sigma_x = 40 \text{ MPa} \\ \sigma_y = -125 \\ \tau_{xy} = 20 \text{ MPa} \end{array} \right.$

$$\sigma_1 = 42.3 \text{ MPa}$$

$$\sigma_2 = -127.3 \text{ MPa}$$

