## Chapter-5 Accelerated Fluid

When a fluid mass is moving with constant acceleration, we assume no relative motion between the fluid layers, i.e. no shear stress.

## 1- Linear motion with constant acceleration.

Assume a fluid in a vessel (of unit width), the vessel is moving with constant acceleration.


Equation of Newton $2^{\text {nd }}$ law in x-direction

$$
\begin{align*}
& m a_{x}=\sum F_{x} \\
& d_{m} a_{x}=P d_{y}-\left(P+\frac{\partial P}{\partial x} d x\right) d y \\
& d_{m} a_{x}=-\frac{\partial P}{\partial x} d x d y \\
& \therefore a_{x}=-\frac{1}{\rho} \frac{\partial P}{\partial x} \tag{1}
\end{align*}
$$

Equation of Newton $2^{\text {nd }}$ law in y-direction

$$
\begin{align*}
& m a_{y}=\sum F_{y} \\
& d_{m} a_{y}=P d_{x}-\left(P+\frac{\partial P}{\partial y} d y\right) d x-g d m \\
& d_{m} a_{y}=-\frac{\partial P}{\partial y} d x d y-g d m \\
& d m=\rho(d x * d y * 1) \\
& \therefore a_{y}=-g-\frac{1}{\rho} \frac{\partial P}{\partial y} \tag{2}
\end{align*}
$$

Note: if $a_{y}=0$, the pressure along y direction will vary hydrostatically i.e. $\mathrm{P}=\gamma \mathrm{h}$.
But, $d P=\frac{\partial P}{\partial x} d x+\frac{\partial P}{\partial y} d y$
Hence, from equations (1) and (2),

$$
\begin{equation*}
d P=-\rho a_{x} d_{x}-\left(\rho g+\rho a_{y}\right) d y \tag{3}
\end{equation*}
$$

The line of constant pressure, can be found from the above equation, by setting $d P=0$

$$
\Rightarrow \rho a_{x} d_{x}=-\rho\left(g+a_{y}\right) d y
$$

$\therefore \frac{d y}{d x}=-\frac{a_{x}}{g+a_{y}} \quad$ (negative slope).
The line of constant pressure is free surface itself.

## 2- Rotation with constant acceleration

## Assumptions:

- No pressure variation with $\theta$ direction
- The horizontal rotation will not alter the pressure distribution in the vertical direction (i.e. the pressure equals to $P=\gamma h$ ).

Applying Newton's $2^{\text {nd }}$ low in r-direction:
$-m a_{r}=\sum F_{r}$

$-d m a_{r}=\sum F_{r}$
$-\rho r d \theta d r d z a_{r}=\operatorname{Pr} d \theta d z-\left(P+\frac{\partial P}{\partial r} d r\right) d z r d \theta$
$\therefore \frac{\partial P}{\partial r}=\rho a_{r}$
$a_{r}=r \omega^{2}$
$\therefore \frac{\partial P}{\partial r}=\rho r \omega^{2}$
$-m a_{z}=\sum F_{z}=0, a_{z}=0$
$P r d \theta d r-\left(P+\frac{\partial P}{\partial z} d z\right) r d r d \theta-\rho r d r d \theta d z g=0$
$\therefore \frac{\partial P}{\partial z}=-\rho g$
But, $d P=\frac{\partial P}{\partial r} d r+\frac{\partial P}{\partial z} d z$
$d P=\rho r \omega^{2} d r-\rho g d z$
on the free surface, $\mathrm{dP}=0$.

$$
\omega^{2}\left(\frac{r_{2}^{2}}{2}-\frac{r_{1}^{2}}{2}\right)=g\left(z_{2}-z_{1}\right)
$$

If we put point 1 at the z -axis so that $r_{1}=0$
$\omega^{2} \frac{r_{2}^{2}}{2}=g\left(z_{2}-z_{1}\right)$ Equation of Parabola.

## Example 5.1

The tank shown in Fig. 1a is accelerated to the right. Calculate the acceleration ax needed to cause the free surface shown in Fig. 1b to touch point A. Calculate also the pressure at point B.


## Example 5.2

A closed box with horizontal base of $6 \times 6 \mathrm{~m}$ and height of 2 m is half filled with water. It is given $a_{\mathrm{x}}=\mathrm{g} / 2$ and $a_{\mathrm{y}}=-\mathrm{g} / 4$. Find the pressure at point b as shown.


## Example 5.3

A water-filled cylinder is rotating about its center line. Calculate the rotational speed that is necessary for the water to just touch the origin and the pressures at A and B.


