

Fin efficiency

$$\eta = \frac{q \text{ actual}}{q \text{ max}}$$

Applied this equation on case 1

$$\eta = \frac{(hpKA)^{0.5} (T^{\circ} - T_{\infty})}{hpL (T^{\circ} - T_{\infty})} = \frac{1}{mL}$$

For case 2

$$\eta = \frac{(hpKA)^{0.5} (T^{\circ} - T_{\infty}) \tanh(mL)}{hpL (T^{\circ} - T_{\infty})} = \frac{\tanh(mL)}{mL}$$

The value of η may be found from figure 1 or figure 2

Figure 1 Efficiencies of straight rectangular and triangular fins.

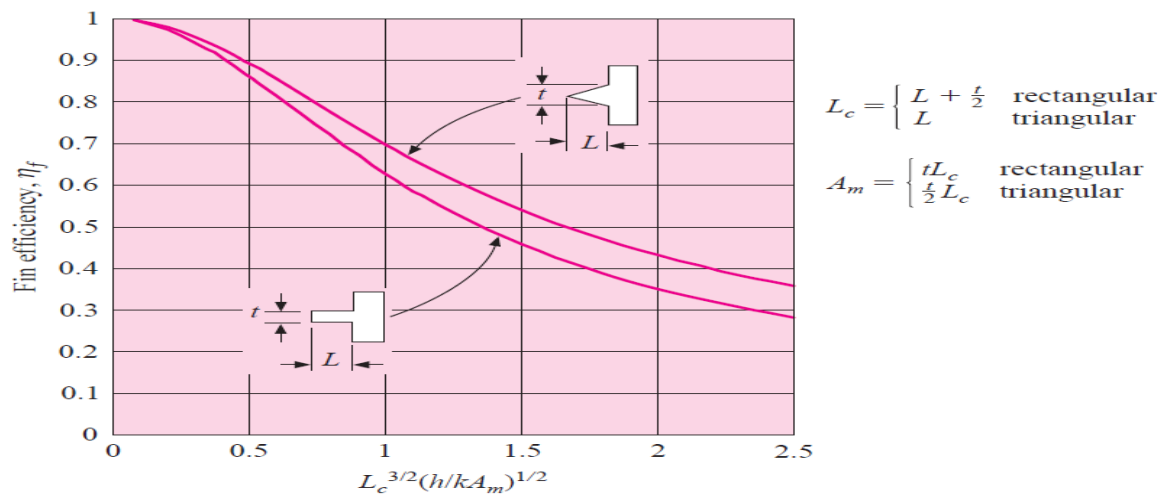
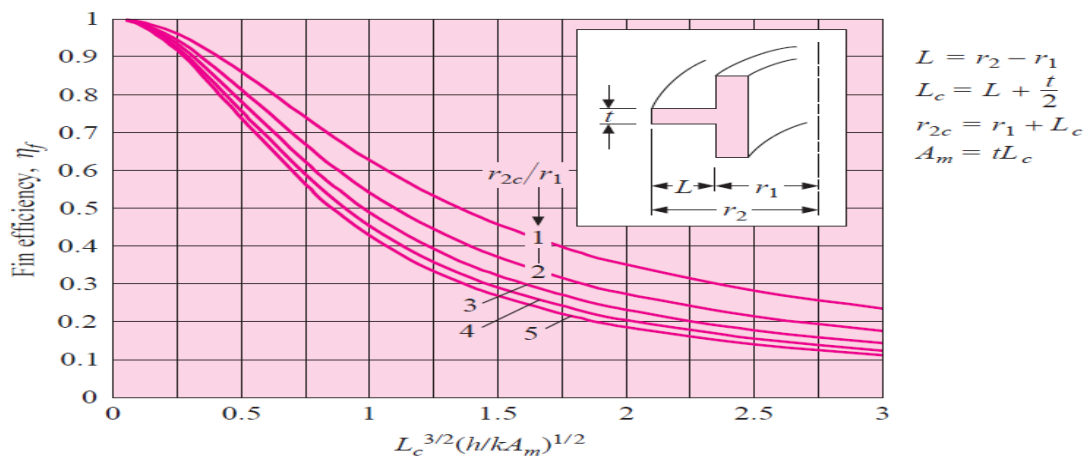


Figure 2 Efficiencies of circumferential fins of rectangular profile.



$$q_{\max} \text{ for circumferential fins} = 2\pi h(r_2^2 - r_1^2) (T^{\circ} - T_{\infty})$$

$$q_{\max} \text{ for rectangular fins} = hpL\theta^{\circ}$$

Fin performance

In some cases a valid method of evaluating fin performance is to compare the heat transfer with the fin to that which would be obtained without the fin. The ratio of these quantities

$$\frac{q_{\text{with fin}}}{q_{\text{without fin}}} = \frac{\eta_f A_f h \theta^{\circ}}{h A_b \theta^{\circ}}$$

where A_f is the total surface area of the fin and A_b is the base area.

A_f = Area of fin

A_b = Area of base

The heat ratio become in case 2

$$\frac{q_{\text{with fin}}}{q_{\text{without fin}}} = \frac{\tanh ml}{\left(\frac{hA}{kp}\right)^{0.5}}$$

The ratio of heat called fin effectiveness

Example:

Aluminum fins 1.5 cm wide and 1.0 mm thick are placed on a 2.5cm diameter tube to dissipate the heat. The tube surface temperature is 170°C, and the ambient-fluid temperature is 25°C. Calculate the heat loss per fin for $h = 130 \text{ W/m}^2 \cdot ^{\circ}\text{C}$. Assume $k = 200 \text{ W/m} \cdot ^{\circ}\text{C}$ for aluminum.

Solution

$$L_c = L + t/2 = 1.5 + 0.05 = 1.55 \text{ cm}$$

$$r_1 = 2.5/2 = 1.25 \text{ cm}$$

$$r_2 = r_1 + L_c = 1.25 + 1.55 = 2.80 \text{ cm}$$

$$r_2/r_1 = 2.80/1.25 = 2.24$$

$$A_m = t \cdot L_c = (0.001)(1.55)(10^{-2}) = 1.55 \times 10^{-5} \text{ m}^2$$

$$L_c^{3/2} \left(\frac{h}{k A_m}\right)^{1/2} = (0.0155)^{3/2} \left(\frac{130}{200 \cdot 1.55 \times 10^{-5}}\right)^{1/2} = 0.369$$

From Figure 2, $\eta_f = 82$ percent

$$q_{\max} = 2\pi(r_2^2 - r_1^2)(h) (T^{\circ} - T_{\infty})$$

$$= 2\pi(2.8^2 - 1.25^2)(10^{-4})(130)(170 - 25) = 74.35 \text{ W}$$

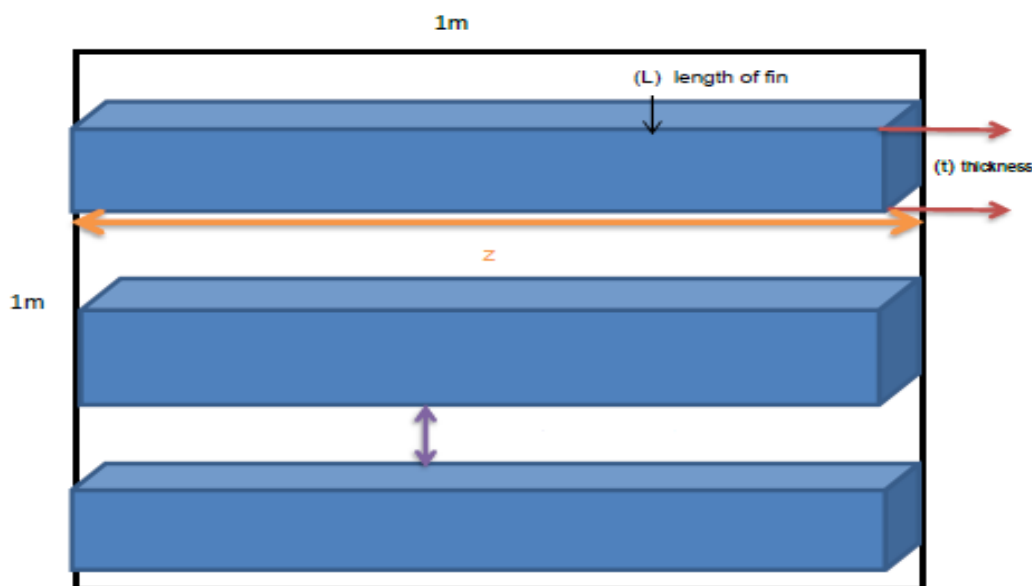
$$\eta = \frac{q_{\text{actual}}}{q_{\text{max}}} \quad q_{\text{actual}} = (0.82)(74.35) = 60.97 \text{ W}$$

Example:

Calculate the heat loss from 250 rectangular thin fins of uniform cross section of thickness 0.5 mm and height 50 mm on (1*1)m surface ,separated by a distance of 4 mm .The heat transfer coefficient is 30 w/m² °C and the thermal conductivity is 200 W/m °C.

Solution:

$$q_{\text{with fins}} = \text{no. of fin} * q_{\text{fin}} + q_{\text{between fins}}$$



$$q_{\text{fin}} = k A m \theta^{\circ} \tanh (mL_c)$$

$$L_c = L + t/2 = 5 + 0.025 = 5.025 \text{ cm}$$

$$m = \sqrt{\frac{2h}{kt}} = \sqrt{\frac{2*30}{200*0.5*10^{-3}}} = 24.5$$

$$m L_c = 24.5 * 5.025 * 10^{-2} = 1.23$$

$$q_{\text{fin}} = 200 * 24.5 * (1*0.5 * 10^{-3}) * \theta^{\circ} * \tanh(1.23) = 2.06 \theta^{\circ}$$

$$q_{\text{between fin}} = h A_{\text{between fins}} \theta^{\circ}$$

$$A_{\text{between fins}} = 1(1 - (0.5 * 10^{-3} * 1 * 250)) = 0.875 \text{m}^2$$

$$q_{\text{between fins}} = 30 * 0.875 * \theta^{\circ} = 26.25 \theta^{\circ}$$

$$q_{\text{with fins}} = 250 * 2.06 \theta^{\circ} + 26.25 \theta^{\circ} = 541.25 \theta^{\circ}$$

*Calculate the percentage of increase of heat transfer rate due to addition of number of rectangular fins ,if the heat transfer coefficient is $40 \text{w/m}^2 \text{ }^{\circ}\text{C}$ before addition the fins

$$q_{\text{with fins}} = 541.25 \theta^{\circ}$$

$$q_{\text{without fins}} = h A_b \theta^{\circ} = 40 * (1 * 1) * \theta^{\circ} = 40 \theta^{\circ}$$

$$\frac{q_{\text{with fins}}}{q_{\text{without fins}}} = \frac{541.25 \theta^{\circ}}{40 \theta^{\circ}} = 13.53$$